

SPECIFICATION

To All Whom It May Concern:

Be It Known That, **David Nelson**, a citizen of the United States, resident of the City of Chicago, in the State of Illinois, whose full post office address is 5834 North Drake Avenue, Chicago, Illinois 60659, and **William Stern**, a citizen of the United States, resident of the City of Dyer in the State of Indiana, whose full post office address is 161 Valleyview Lane, Dyer, Indiana 46311 have invented certain new and useful improvements in:

UNIVERSAL JOINT WASHER BAFFLE

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] Not Applicable.

STATEMENT REGARDING FEDERALLY SPONSORED RESEARCH

[0002] Not Applicable.

BACKGROUND OF THE INVENTION

[0003] The present invention relates generally to universal joints having cross-member journals provided with internal lubrication channels which open at the ends of the journals to provide for replenishment of lubricant in bearing cups within which the journals are pivotally supported in respective yoke members, and in particular, to a washer and lubrication baffle configured for press-fitting retention within an enlarged end portion of the lubrication channel.

[0004] Typically, within the cross-members or trunnions of a universal joint, the lubrication channels intersect at a central location, adjacent an external lubrication fitting, whereby lubricant can be introduced to the universal joint when required, as shown in U.S. Patent No. 4,103,512 to *McElwain, et al.* The lubricant introduced to the central location of the universal joint typically flows radially outward towards the ends of the cross-member journals through the individual lubrication channels. During operation, the rotational movement of the universal joint about an axis perpendicular to the cross-members generates centrifugal forces which assists the outward flow of lubricant to the bearing cups.

[0005] When rotation of the universal joint is stopped, one or more of the journals may be disposed in a vertically upward orientation, such that the force of gravity acts to draw lubricant back downward from the bearing cups and into the lubrication channels. Upon

resumption of rotational motion, the associated bearing caps may be left with little or no lubricant, resulting in higher frictional forces and undesired wear.

[0006] In addition to the use of lubricant to reduce frictional wear between the cross-member ends and bearing caps, manufacturers often dispose a thrust washer as a buffer between the inner surface of the bearing cap and the cross-member or trunnion end, such as is shown in U.S. Patent No. 6,264,566 B1 to *Nieman et al.*, or in U.S. Patent No. 5,813,916 to *Lentini, et al.* During manufacture, these thrust washers are installed in the bearing caps prior to placement over the cross-member ends, using spacers to ensure proper location. The corresponding faces of the cross-member ends are ground in a machining operation to ensure a desired tolerance for the thrust washers, and are pressed to provide for one or more lubrication grooves facilitating a flow of lubricant between the thrust washer and the cross-member end.

[0007] Accordingly, it is an object of the present invention to overcome the potential problems associated with lubricant drainage from the proximity of the bearing cups in a universal joint while simultaneously eliminating one or more grinding and machining steps from the universal joint manufacturing process.

BRIEF SUMMARY OF THE INVENTION

[0008] Briefly stated, the present invention provides a combination of a thrust washer and lubrication baffle adapted for a press fitting installation into an enlarged lubrication passage of a universal joint cross-member or trunnion. The thrust washer and unitary lubrication baffle incorporates one or more lubrication grooves, facilitating the flow of lubricant between the surfaces of the cross-member and bearing cap. Elastomeric properties of the thrust washer eliminate the need for grinding of the cross-member

ends to desired tolerances, and the incorporation of lubrication grooves into the thrust washer surface eliminates the need for pressing of lubrication grooves directly into the cross-member ends prior to hardening or heat treating.

[0009] The foregoing and other objects, features, and advantages of the invention as well as presently preferred embodiments thereof will become more apparent from the reading of the following description in connection with the accompanying drawings.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

[0010] In the accompanying drawings which form part of the specification:

[0011] Figure 1 is a plan view of a prior art universal joint cross member;

[0012] Figure 2 is a side sectional view of a washer baffle of the present invention;

[0013] Figure 3 is a top plan view of the washer baffle of Fig. 2;

[0014] Figure 4 is a bottom plan view of the washer baffle of Fig. 2;

[0015] Figure 5 is an enlargement of a portion of Fig. 4, illustrating a vertically aligned rib for press-fit attachment to a cross-member;

[0016] Figure 6 illustrates the placement of a washer baffle of Fig. 2 in a cross member;
and

[0017] Figure 7 is a sectional view of a washer baffle of Fig. 2 press fitted to a cross-member of a universal joint.

[0018] Corresponding reference numerals indicate corresponding parts throughout the several figures of the drawings.

DESCRIPTION OF THE PREFERRED EMBODIMENT

[0019] The following detailed description illustrates the invention by way of example and not by way of limitation. The description clearly enables one skilled in the art to make

and use the invention, describes several embodiments, adaptations, variations, alternatives, and uses of the invention, including what is presently believed to be the best mode of carrying out the invention.

[0020] Turning to Figure 1, a conventional Hooke-type universal joint cross-member or trunnion is shown generally at 10. The cross-member 10 consists of four equidistantly spaced journals 12 disposed about a central body 14. A lubricant channel 16 extends axially through each journal 12, from a common central point C, to an open-ended lubrication chamber 18 adjacent a radially outward end 20 of each journal 12. At the central point C in the central body 14, one or more lubricant access passages 22 are provided for the introduction of a lubricant into the lubricant channels 16.

[0021] Each lubrication chamber 18 is defined by an enlarged diameter portion of the associated lubrication channel 16 having a generally frustoconical radially inner surface 24 receiving the lubrication channel 16, and cylindrical sidewalls 26. The radially outer end of the lubrication chamber 18 is open, and includes a chamfered peripheral edge 28 transitioning between the cylindrical sidewalls 26 and the outward end 20 of the journal 12.

[0022] Turning to Figures 2-4, an washer baffle combination of the present invention is shown generally at 100. The washer baffle consists of a thrust washer portion 102, and an associated baffle portion 104 adapted for press-fitting into the lubrication chamber 18 of a universal joint journal 12. Preferably, the thrust washer portion 102 and the associated baffle portion 104 are of a unitary elastomeric construction. One suitable material from which the washer baffle combination may be formed using conventional plastic forming techniques is a nylon elastomer identified as DuPont 70G33L, and sold

under the "Zytel" brand name by E. I. duPont de Nemours & Co. of Wilmington, Delaware. Those of ordinary skill in the art will recognize that other materials, not limited to elastomers, may be utilized in the construction of the washer baffle combination 100 of the present invention, provided that necessary tolerances for wear, temperature, and elasticity are met.

[0023] As best seen in Figure 3, the thrust washer portion 102 of the washer baffle 100 defines an annular surface substantially corresponding to the dimensions of the radially outward end 20 of a journal 12. Disposed in at least the upper surface of the thrust washer portion 102 are a plurality of equidistantly spaced lubricant grooves 106 radiating outward from a central axis X of the thrust washer portion 102. Optionally, lubricant grooves 106 may be provided in the lower surface of the thrust washer portion 102 as well.

[0024] Associated with the thrust washer portion 102, the baffle portion 104 is disposed concentrically about the central axis X, perpendicular to the lower surface of the thrust washer portion 102. The baffle portion 104 is defined by a pair of concentric cylindrical elements 108A and 108B joined together by an annular base 110. Cylindrical element 108A defines the outer cylindrical surface of the baffle portion 104, and is preferably contiguous with the thrust washer portion 102. As best seen in Figure 5, the outer surface of the cylindrical element 108A preferably includes a plurality of ribs 111 aligned parallel to the central axis X. Each rib 111 is configured to provide a press fit engagement with an inner surface of the journal lubrication chamber 18. Cylindrical element 108B defines an inner cylindrical surface of the baffle portion 104, as well as an open ended axial lubrication delivery passage 112. The annular base 110 joins the

concentric cylindrical elements 108A and 108B opposite the thrust washer portion 102, defining an annular lubrication retaining volume 114 concentric with the axial lubrication delivery passage 112. Preferably, the annular base 110 and the cylindrical element 108B are contiguous with the cylindrical element 108A, however, those of ordinary skill in the art will recognize that the components of the washer baffle combination 100 may be formed discretely and joined in conventional manners to provide the equivalent functionality of contiguously formed components.

[0025] Turning to Figures 6 and 7, a press-fit insertion of the thrust washer baffle combination 100 axially into the lubrication chamber 18 of a journal 12 is shown. As the thrust washer baffle combination 100 is press-fitted to the lubrication chamber 18, ribs 111 ensure sufficient engagement between the outer surface of the cylindrical element 108A and the surfaces of the lubrication chamber 118, retaining the thrust washer baffle combination therein in a centered position. Upon complete insertion into the lubrication chamber 118, the underside of the thrust washer portion 102 is seated against the journal outward end 20, and the axial lubrication delivery passage 112 is disposed coaxially with the lubrication channel 16 to receive a flow of lubricant therefrom. Elastomeric properties of the thrust washer portion 102 function to provide a limited range of tolerance for dimensional variations in the journal 12 and journal outward end 20, eliminating the need to grind surfaces of the journal 20 to precise tolerances during a manufacturing process.

[0026] Once the thrust washer baffle combination 100 is seated in the lubrication chamber 18 of a journal 12, as shown in Figure 7, a bearing cup 200 is fitted over the thrust washer portion 102 and journal outward end 20. The bearing cup 200 is retained

in a limb 202 of a joint yoke by a snap ring 204. Needle roller bearings 206 are disposed between the outer circumferential surface of the journal 12 and the inner circumferential side wall of the bearing cup 200. An annular seal 208 in a carrier 210 retains lubricant and excludes contaminants from the inside of the bearing cap 200.

[0027] During operation of a universal joint including one or more thrust washer baffle combinations 100 of the present invention, lubricant flows radially outward from the center of the universal joint cross-member 10 along the respective lubrication channels 16, towards the journal outer ends 20. When the lubricant flow enters the lubrication chamber 18, outward flow is restricted by the annular base 110 to flow through the open ended axial lubrication delivery passage 112, where it is delivered to the journal outer end 20. From the journal outer end 20, the lubricant flow continues between the various surfaces of the thrust washer portion 102, along the equidistantly spaced lubricant grooves 106 to provide lubrication to the needle bearings 206 and other wear surfaces. When rotational motion of the universal joint stops and the flow of lubricant reverses, such as due to the influence of gravity, a volume of lubricant is retained in the annular volume 114, and prevented from returning to the lubricant channel 16, thereby ensuring an adequate lubricant supply at the journal outer end 20 upon a resumption of rotational motion.

[0028] In view of the above, it will be seen that the several objects of the invention are achieved and other advantageous results are obtained. As various changes could be made in the above constructions without departing from the scope of the invention, it is intended that all matter contained in the above description or shown in the accompanying drawings shall be interpreted as illustrative and not in a limiting sense.